

**MANNESMANN  
REXROTH****Fixed Displacement Motor A4FM**Series 3, for open and closed circuits  
axial piston - swashplate design**RE  
91100/12.95**

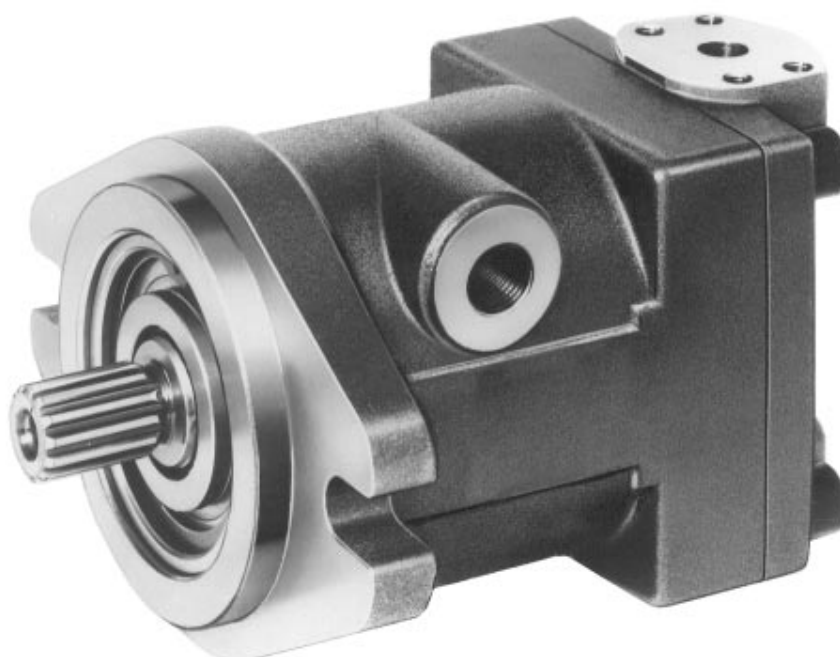
Brueninghaus Hydromatik

Sizes 22...56

Nominal pressure 400 bar

Peak pressure 450 bar

replaces 10.90



Fixed displacement motor A4FM is an axial piston motor of swashplate design suitable for use in both open and closed circuits in high pressure ranges.

Output speed is proportional to input flow and inversely proportional to motor displacement. Output torque increases with the pressure drop across the motor between the high and low pressure sides.

This motor is suitable for both industrial and mobile applications.

**Special Features**

- Proven A4 rotary group
- Long service life
- Favourable power/weight ratio
- Compact design for special applications where A2FM cannot be applied
- Optimum efficiencies
- Economic design
- SAE mounting flange



Fixed Displacement Motor A4FM

**Ordering Code**

**A4F M / 3 1 W - P C**

**Hydraulic fluid**

Mineral oil (no code)

**Axial piston unit**

Swashplate design, fixed displacement **A4F**

**Mode of operation**

Motor **M**

**Size**

22 28 40 56

≅ Displacement  $V_g$  (cm<sup>3</sup>) ● ● ● ●

**Series**

3

**Index**

1

**Direction of rotation**

Viewed on shaft end alternating **W**

**Seals**

NBR (nitril-caoutchouc) **P**

**Shaft end**

22 28 40 56

Splined shaft SAE 7/8"	●	●	-	-	<b>S</b>
Splined shaft SAE 1"	-	●	-	-	<b>T</b>
Splined shaft DIN 5480	-	-	●	●	<b>Z</b>

**Mounting flange**

SAE 2-hole ● ● ● ● **C**

**Service line connections**

Ports A and B SAE at side (opposite side)	●	●	●	-	<b>02</b>
Ports A and B SAE at side (same side)	-	-	-	●	<b>10</b>

- = available
- = not available

## Technical Data

### Fluid

We request that before starting a project detailed information about the choice of pressure fluids and application conditions are taken from our catalogue sheets RE 90220 (mineral oil), RE 90221 (environmentally acceptable hydraulic fluids) and RE 90223 (fire resistance fluids, HF).

When using HF- or environmentally acceptable hydraulic fluids possible limitations for the technical data have to be taken into consideration. If necessary please consult our technical department (please indicate type of the hydraulic fluid used for your application on the order sheet).

### Operating viscosity range

In order to obtain optimum efficiency and service life, we recommend that the operating viscosity (at operating temperature) be selected from within the range:

$$v_{\text{opt}} = \text{operating viscosity } 16 \dots 36 \text{ mm}^2/\text{s}$$

referred to the loop temperature (closed circuit) or tank temperature (open circuit).

### Viscosity limits

The limiting values for viscosity are as follows:

$$v_{\text{min}} = 5 \text{ mm}^2/\text{s},$$

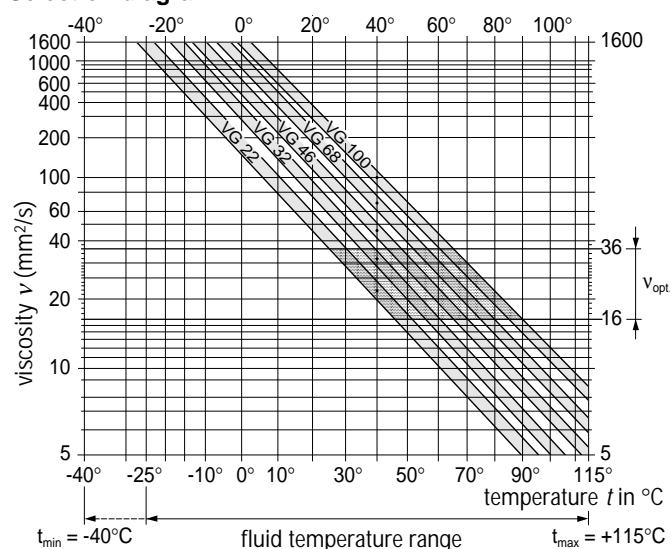
short term at a max. permissible temperature of  $t_{\text{max}} = 115^\circ\text{C}$

$$v_{\text{max}} = 1600 \text{ mm}^2/\text{s}, \text{ short term on cold start } (t_{\text{min}} = -40^\circ\text{C})$$

Please note that the max. fluid temperature of  $115^\circ\text{C}$  is also not exceeded in certain areas (for instance bearing area).

At temperatures of  $-25^\circ\text{C}$  up to  $-40^\circ\text{C}$  special measures may be required for certain installation positions. Please contact us for further information.

### Selection diagram



### Notes on the selection of the hydraulic fluid

In order to select the correct fluid, it is necessary to know the operating temperature in the loop (closed circuit) or the tank temperature (open circuit) in relation to the ambient temperature. The hydraulic fluid should be selected so that within the operating temperature range, the operating viscosity lies within the optimum range ( $v_{\text{opt}}$ ) (see shaded section of the selection diagram). We recommend that the highest possible viscosity range should be chosen in each case.

Example: At an ambient temperature of  $X^\circ\text{C}$  the operating temperature is  $60^\circ\text{C}$ . Within the operating viscosity range ( $v_{\text{opt}}$ ,

shaded area), this corresponds to viscosity ranges VG 46 or VG 68. VG 68 should be selected.

Important: The leakage oil (case drain oil) temperature is influenced by pressure and motor speed and is always higher than the circuit temperature. However, at no point in the circuit may the temperature exceed  $115^\circ\text{C}$ .

If it is not possible to comply with the above condition because of extreme operating parameters or high ambient temperatures we recommend housing flushing. Please consult us.

### Filtration

The finer the filtration the better the achieved purity grade of the pressure fluid and the longer the life of the axial piston unit. To ensure the functioning of the axial piston unit a minimum purity grade of:

9 to NAS 1638

6 to SAE

18/15 to ISO/DIS 4406 is necessary.

At very high temperatures of the hydraulic fluid ( $90^\circ\text{C}$  to max.  $115^\circ\text{C}$ ) at least cleanliness class

8 to NAS 1638

5 to SAE

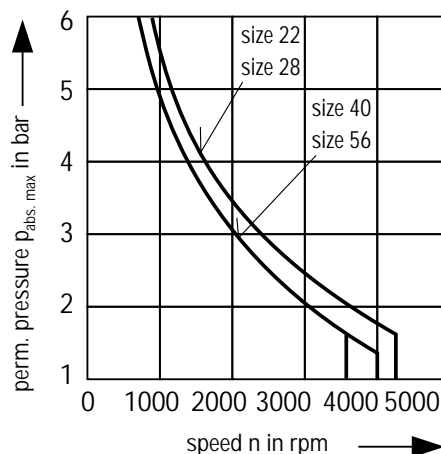
17/14 to ISO/DIS 4406 is necessary.

If above mentioned grades cannot be maintained please consult supplier.

### Case drain pressure

The lower the speed and the case drain pressure the higher the life expectation of the shaft seal ring.

Shaft seal ring **NBR** (nitril-caoutchouc)



The values shown in the diagram are permissible loads of the seal ring and shall not be exceeded.

If the values are exceeded, the life of the shaft seal will be reduced.

Note:

max. permissible speeds of the fixed displacement motor are given in the table on page 4.

max. perm. housing pressure \_\_\_\_\_  $p_{\text{abs,max}} 6 \text{ bar}$ .

The pressure in the housing must be greater than the external pressure on the shaft seal.

### Direction of flow

clockwise rotation

anti-clockwise rotation

**A to B**

**B to A**

## Fixed Displacement Motor A4FM

**Speed range**

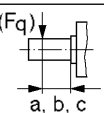
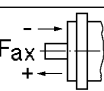
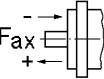
There is no limitation on minimum speed  $n_{min}$ . If uniformity of rotation is required, however, speed  $n_{min}$  should not be allowed to fall below 50 rpm. See table for max. permissible speeds.

**Installation position**

Optional. The motor housing must be filled with fluid prior the commissioning, and must remain full whenever it is operating. For extensive information on installation position, please consult our data sheet RE 90270 before completing your design work.

**Output drive**

permissible axial and radial loading on drive shaft

Size			22	28	28	40	56	
Shaft end			S	S	T	Z	Z	
Distance of $F_q$ (from shaft shoulder) 	a	mm	17,5	17,5	17,5	17,5	17,5	
	b	mm	30	30	30	30	30	
	c	mm	42,5	42,5	42,5	42,5	42,5	
Max. permissible radial force at distance 	a	$F_{q\ max}$	N	1300	1300	2500	3600	5000
	b	$F_{q\ max}$	N	1100	1100	2000	2891	4046
	c	$F_{q\ max}$	N	900	900	1700	2416	3398
Max. permissible axial load 		$\pm F_{ax\ max}$	N	987	987	987	1500	2200
Minimum pinion dia	$D_{R\ min}$	mm	56	56	64	75	75	
Minimum V-belt pulley dia	$D_{K\ min}$	mm	111	111	127	150	150	

**Table of values** (theoretical values, without considering  $\eta_{mh}$  and  $\eta_v$ ; values rounded)

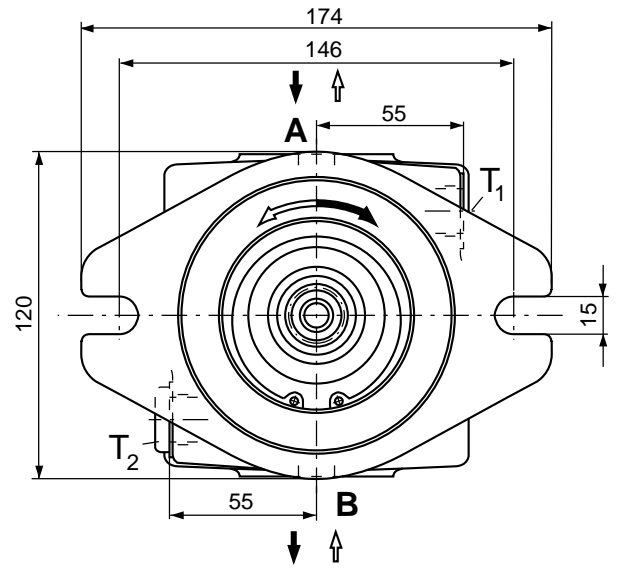
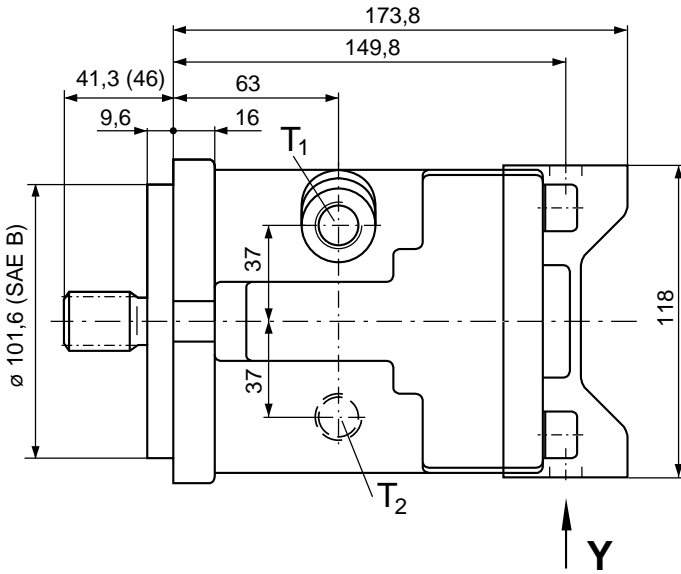
Size			22	28	40	56
Displacement	$V_g$	cm <sup>3</sup>	22	28	40	56
Max. speed	$n_{max\ continuous}$	rpm	4250	4250	4000	3600
Intermittent max. speed <sup>1)</sup>	$n_{max\ interm.}$	rpm	5000	5000	5000	4500
Max. flow	$q_{V\ max}$	L/min	93	119	160	202
Torque constants	$T_K$	Nm/bar	0,35	0,445	0,64	0,89
Torque at $\Delta p = 400$ bar	$T$	Nm	140	178	255	356
Output power at $n_{max}$ u. $\Delta p = 400$ bar	$P$	kW	62	79	107	134
Moment of inertia about drive axis	$J$	kgm <sup>2</sup>	0,0015	0,0015	0,0043	0,0085
Weight (approx.)	$m$	kg	11	11	15	21

<sup>1)</sup> Intermittent max. speed at overspeed:  $\Delta p = 70 \dots 150$  bar

**Calculation of size**

Flow	$q_v = \frac{V_g \cdot n}{1000 \cdot \eta_v}$	in L/min	$V_g$ = geometric displacement per rev.	in cm <sup>3</sup>
			$T$ = torque	in Nm
Output speed	$n = \frac{q_v \cdot 1000 \cdot \eta_v}{V_g}$	in rpm	$\Delta p$ = pressure differential	in bar
			$n$ = speed	in rpm
Output torque	$T = \frac{V_g \cdot \Delta p \cdot \eta_{mh}}{20 \cdot \pi} = \frac{1,59 \cdot V_g \cdot \Delta p \cdot \eta_{mh}}{100}$	in Nm	$T_K$ = torque constants	in Nm/bar
	or $T = T_K \cdot \Delta p \cdot \eta_{mh}$	in Nm	$\eta_v$ = volumetric efficiency	
			$\eta_{mh}$ = mech. -hyd. efficiency	
			$\eta_t$ = overall efficiency	
Output power	$P = \frac{2 \pi \cdot T \cdot n}{60 \cdot 1000} = \frac{T \cdot n}{9549} = \frac{q_v \cdot \Delta p}{600} \cdot \eta_t$	in kW		

**Unit Dimensions, Sizes 22, 28**



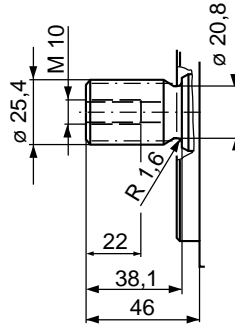
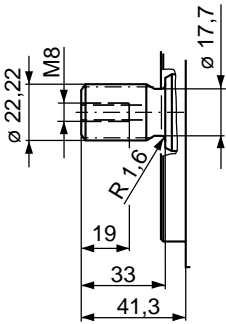
**Shaft ends**

**S** (sizes 22, 28)

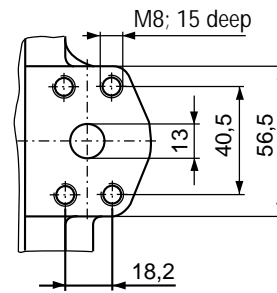
Splined shaft SAE 7/8",  
(SAE B), 30° pressure angle,  
13 teeth, 16/32 pitch,  
flat root, side fit,  
tolerance class 5  
ANSI B92.1a-1976

**T** (sizes 28)

Splined shaft SAE 1",  
(SAE B-B), 30° pressure angle,  
15 teeth, 16/32 pitch,  
flat root, side fit,  
tolerance class 5  
ANSI B92.1a-1976



View Y



**Connections**

A, B Service line ports

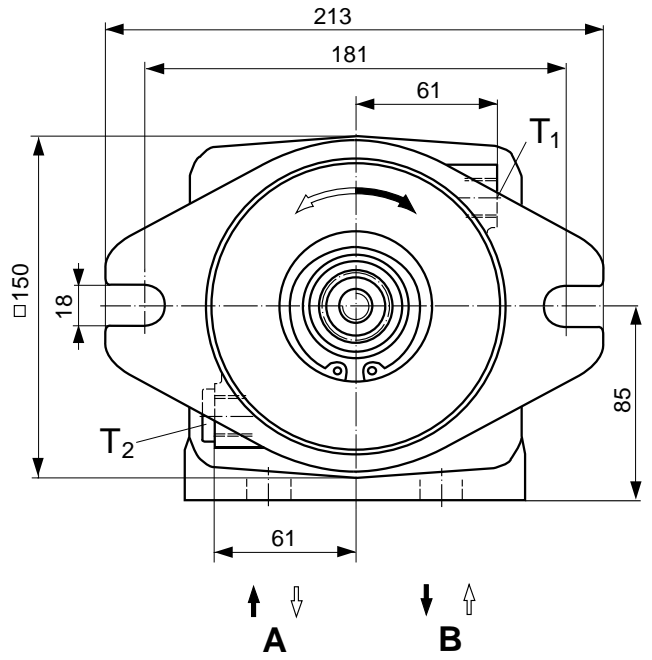
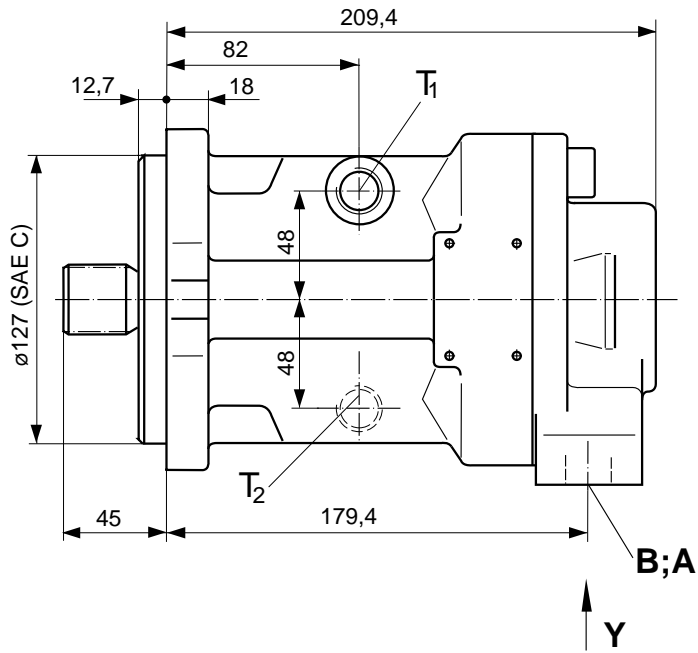
SAE 1/2", 420 bar (6000 psi)  
high pressure series

T1, T2 Leakage port / oil filling port

M18x1,5; 15 deep

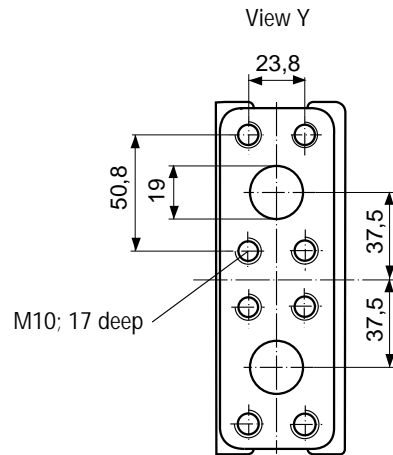
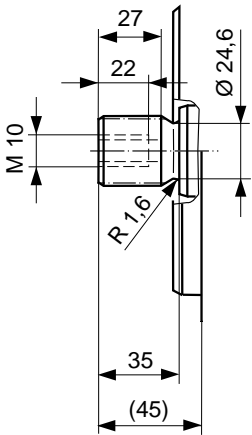


**Unit Dimensions, Size 56**



**Shaft ends**

**Z**  
Splined shaft  
W 30x2x30x14x9g  
DIN 5480



**Connections**

A, B Service line ports

T1, T2 Leakage port / oil filling port

SAE 3/4", 420 bar (6000 psi)  
high pressure series  
M18x1,5; 12 deep

Fixed Displacement Motor A4FM